

WSD General Order No. KCY-5300 & KCY-5301 Item No. 1-4

RATE OF TEMPERATURE CHANGE:

AN UPSET TEMPERATURE OF 670 dg F IS PERMITTED PROVIDED THE
DURATION DOES NOT EXCEED 30 MINUTES PER EXCURSION OF 200 HOURS
IN THE TOTAL LIFE OF THE FAN. BASED ON THIS, THE RATE OF
TEMPERATURE CHANGE IS NOT RESTRICTED PROVIDING THE VIBRATION
LEVELS ARE NONITORED AND ALLOWABLE LEVELS ADHERED TO AS SPECIFIED
ON FOUNDATION DESIGN CRITERIA DWG. 2429D95.

3. CONTRACT PERFORMANCE

FAN LOAD	VOLUME CFM	STATIC PRESSURE			TEMP °F	SPEED RPM	BHP	DENSITY #/FT. ³
		TP INLET	OUTLET	FSP RISE				
TB	1,128,400	-30.5	7.84	38.34	300	954	7415	.0409
NET	992,665	-21.1	5.2	26.30	300	809	4596	.0409
100	933,578	-19.2	4.6	23.80	300	768	3918	.0409
75	754,686	-13.7	2.7	16.40	300	636	2195	.0409
50	555,420	-8.8	2.3	11.10	300	514	1106	.0409
25	324,776	-4.5	0.8	5.30	300	353	325	.0409

Performance based on elevation of _____ feet, 25.20" Hg Bar. Pressure

Performance based on use of Evase expanded to _____ sq. ft.

4. NOISE LEVELS

BAND Center Frequency	H _Z	1 63	2 125	3 250	4 500	5 1000	6 2000	7 4000	8 8000	Overall
Measured at	A	--	--	--	--	--	--	--	--	--
	B									
**		127	120	120	115	114	113	110	105	120
*	C	118	111	108	102	100	98	90	80	120
**	D									
								OVERALL		93

A - Inlet noise, measured _____ feet from inlet

B - Discharge Noise, measured 5 feet from dischargeC - Noise thru housing, measured 5 feet from housing, without inlet noise addedD - Noise around fan, measured 5 feet from housing with inlet noise added, BUT WITHOUT discharge, drive or noise effect of other equipment added.

Above noise levels are:

- * ☒ Sound power Ref. 10^{-12} Watts (metric)
 ** ☒ Sound pressure Ref. .0002 microbars
 ** ☒ A scale ☐ C scale

January 1978

IP7_039125

WSD General Order No. KCY-5300/01/02/03Item No. 1,2,3,4

Comments: _____

5. WHEEL GAUGES/MATERIALS

Hubs: FABRICATED STL. CYLINDER ASTM A36
FLANGE ASTM A441

Center/Backplate: 1.75 " Thick; ASTM A514 GR E

Blades:

Single Thickness:

Blades: _____ " Thick; _____

Blade Liner: _____ " Thick; _____

Reinforcing Rings; _____ " Thick; _____

Corner Pads; _____ " Thick; _____

Airfoil:

Skins: .25 " Thick; ASTM A514 GRA

Ribs: .25 " Thick; ASTM A514 GRA
DIA

Nose: 1.94 " Thick; ASTM A514 GRE

Liner: .25 " Thick; ASTM A514 GRA

Side Plates: .75 " Thick; ASTM A514 GRA

When Field Balancing of Rotor is necessary follow procedures outlined in Section 16.2 - Field Balancing of Heavy Duty Fan Wheels. Use Category III for welding weights to wheel.

6. BEARINGS: Size 12 " Dia; Type (W) HD SLEEVE

COOLING MEDIUM:

☐ Water: _____ GPM Max; _____ GPM Min.
 Water Temperature; _____ °F Max; _____ °F Min.
 Water Pressure; _____ PSI Max; _____ PSI Min.

January 1978

IP7_039126

WSD General Order No. KCY-5300/01/02/03 Item No. 1, 2, 3, 4☐ Air: _____ CFM at _____ "H₂O Static Pressure

Air Temperature; _____ of Max; _____ of Min.

☐ Non-Cooled (Ambient Air)☒ External LubricatorLUBRICANT☐ Grease: Type _____☒ Oil: Viscosity 300 SSU at 100 °F
(See Section 6 for complete Oil description)

Quantity: _____ Qts. per Bearing - Self Contained

12 GPM Max.; 8 GPM Min. per Bearing - Flood LubricatedLUBRICATING METHOD☐ Ring - Self Contained☐ Disc - Self Contained☒ Flood Lubricated - From External LubricatorBEARING OPERATING TEMPERATURE

Babbitt: Per Tip Sensitive Thermocouple or Resistance Temperature Detector

140 °F to 160 °F Normal Operating Range180 °F Alarm200 °F Shutdown

Oil temperature for Start-up is 50°F for Self-contained Bearings; 90°F for Flood Lubricated Bearings.

7. LUBRICATOR: Nominal Size (Tank Capacity) 150 GallonsHEAT EXCHANGERCIRCUIT☐ Water-to-Oil☐ Single☒ Air-to-Oil☒ Dual

WSD General Order No. KCY-5300/01/02/03 Item No. 1, 2, 3, 4

Electrical Requirements:

Pump Motors: 5 HP, 1800 RPM3 Phase, 60 HZ, 230/460 Volts

_____Reservoir Heater: 9.25 KW, 3 Phase 60 HZ 480 VoltsAir Cooled Heat Exchanger Fan Motor: 1.5 HP 1200 RPM,3 Phase, 60 HZ, 230/460 Volts

_____8. TURNING GEAR

Turning Gear Assembly designed to:

☐ Start fan from Rest☐ Engage during coast down ONLY and maintain specified RPM.

Motor: _____ HP, _____ RPM

3 Phase, _____ HZ, _____ Volts

Gear: Output RPM _____ (Input to Fan)

Motor-Gear Designed for:

Fan Rotor: Weight _____ Lbs., WR^2 _____ Lbs/Ft.²Drive Rotor: Weight _____ Lbs., WR^2 _____ Lbs/Ft.²

Bearing; Dia. _____"; Sleeve Length _____"

January 1978

IP7_039128

WSD General Order No. KCY-5300 & KCY-5301 Item No. 1-4**9. DRIVE**

Main Drive: ☐ Motor _____ HP _____ RPM
3 Phase, _____ H_Z _____ Volts
☐ Single Speed; ☐ Two-Speed; ☐ Variable Speed
☐ Turbine _____ HP _____ RPM at Test Block
_____ RPM Maximum Overspeed

Overspeed: Continuous ☐Momentary ☐

Secondary Drive: ☐ Motor _____ HP, _____ RPM
3 Phase, _____ H_Z _____ Volts
☐ Turbine _____ HP, _____ RPM at Design
_____ RPM at Max. Overspeed

Overspeed: Continuous ☐Momentary ☐

Variable Speed Drive: Type: _____
(Fluid Drive, Magnetic Coupling, etc)

Make: _____

Model: _____ HP _____

RPM - Min. _____ Max. _____

10. VIBRATION AMPLITUDES

The following table indicates the normal and allowable horizontal and vertical vibration levels of our products. Also shown are levels at which corrective action is required. All values shown are in mils; peak-to-peak amplitude as measured on the bearing housing, horizontal centerline with a seismic measuring device.

January 1978

IP7_039129

WSD General Order No. KCY-5300/01/02/03 Item No. 1, 2, 3, 4

Operating Speed RPM	Peak-to-Peak Amplitude - Horizontal and Vertical - Mils		
	Normal	Rough Alarm	Correction Required Shutdown
1800	0 to 2.0	3.5	5.0
1500	0 to 2.0	4.0	5.5
1200	0 to 2.5	4.5	7.0
1000	0 to 3.0	5.5	7.5
900	0 to 3.0	6.0	8.5
750	0 to 3.5	7.0	9.5
720	0 to 3.5	7.0	9.5
600	0 to 4.5	8.0	11.0
514	0 to 5.0	8.5	12.5

NOTE: For allowable Axial vibration levels, use one-half of above values.

11. SUPPLEMENTAL DRAWINGS

<u>DRAWING NO.</u>	<u>DESCRIPTION</u>	<u>SECTION</u>
<u>2090F87</u>	<u>CONTRACT DWG</u>	<u>5</u>
<u>7644A41</u>	<u>HARDWARE LIST</u>	<u>5</u>
<u>2433D04</u>	<u>OUTLET DAMPER SEAL AIR FAN</u>	<u>5</u>
<u>5453C50</u>	<u>BRACING ROD</u>	<u>5</u>
<u>5454C88</u>	<u>CASING CUT FOR ROTOR REMOVABLE</u>	<u>5</u>
<u>7644A48</u>	<u>THERMOCOUPLE</u>	<u>6</u>
<u>5910C29</u>	<u>BEARING ACCESSORIES</u>	<u>7</u>

January 1978

IP7_039130

Section 5.3 Identification of Parts



LB 90-400-5.3.0 1

The Contract Drawing shows Pictorially, by Bill of Material Legend or notes, the fan with all features and accessories included in the contract. In general, fan components are shipped separately, and Housing and Inlet Box assemblies are shipped in several pieces. Therefore Field Identification of the components and pieces for a specific fan can be complicated. Refer to Figure 5.3-1 for the basic fan

and components and their relative positions. The following paragraphs indicate the ways that components and pieces have been marked at the factory for simplified Field Identification.

Additional care must be exercised in identifying components and pieces on orders with multiple units of

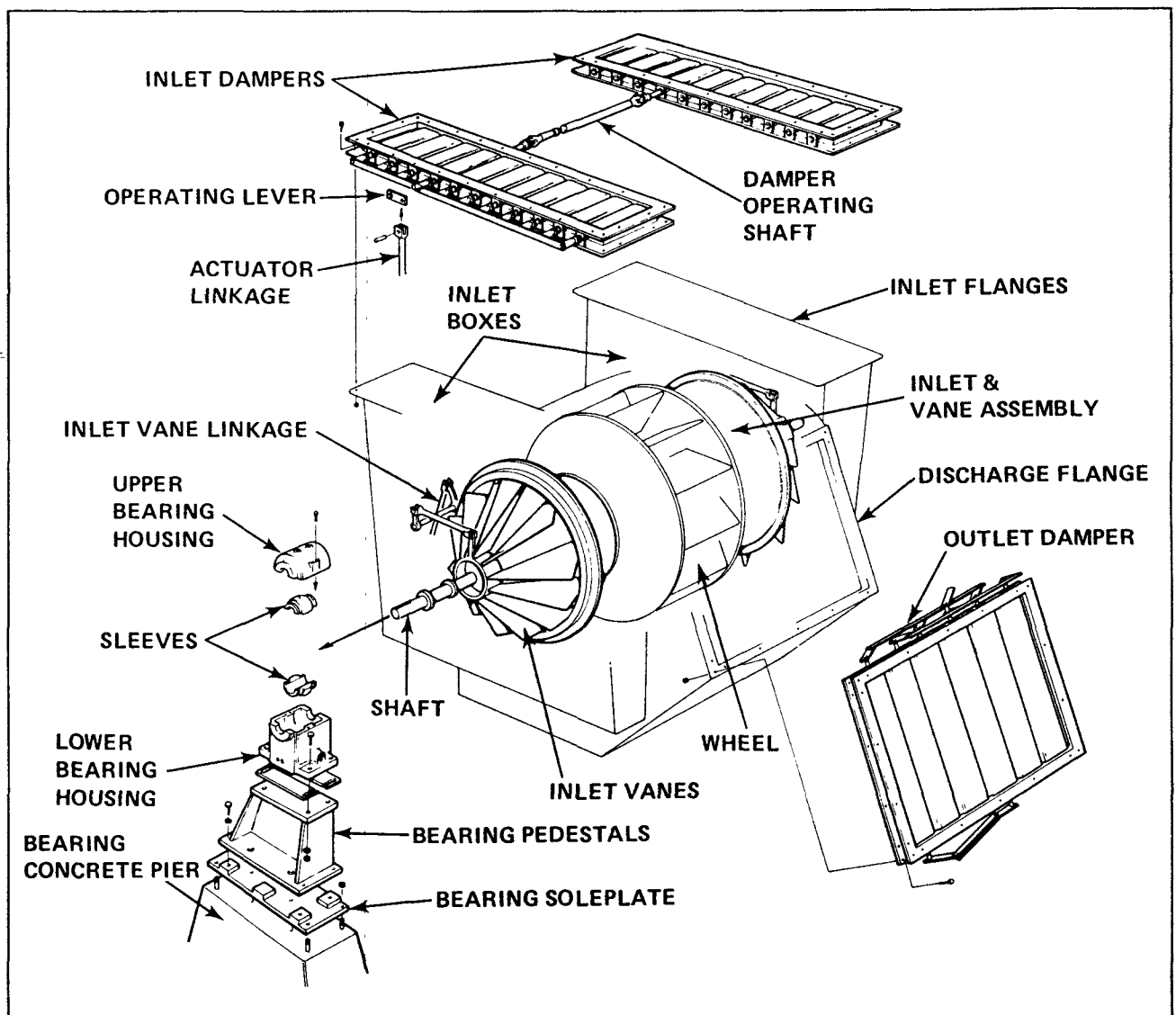


Fig. 5.3-1 Fan Components and Accessories

Effective October

Westinghouse Electric Corporation

Sturtevant Division
Hyde Park, Boston, MA 02136

IP7_039132

5.5.4.14

S. Check complete Housing/Inlet Box alignment in relation to Rotor and all center lines. Verify that Side Sheets are vertical. Correct as required.

2. INLET INSTALLATION AND ALIGNMENT

NOTE

- Rotor must be leveled and aligned to Drive Train Centerline and at proper elevation.
- Inlets must be aligned to Wheel as shown in the Wheel and Inlet Relationship detail on Contract/Main Assembly Drawing. See Figure 5.5.3-14 for typical Wheel and Inlet relationship details. Figure 5.5.3-13 shows the Rotor and Housing/Inlet Box in cross section.
- The Wheel-to-Inlet relationship detail on Contract Drawing will show the cold or ambient temperature setting. Fans for elevated temperature applications will have a clearance at the top of Wheel greater than at the bottom in the cold condition. The Wheel and Inlet will come into alignment at operating temperature.
- Vane and Inlet Assemblies must be positioned with the Operating Lever and Ring Assembly in the position indicated on Contract/Main Assembly Drawing.
- Final axial adjustment on Series 2300 and 2400 Fans must be made by loosening machine screws holding the Sliding Seal Ring to the Fabricated Steel Inlet, and sliding the Seal Ring to the correct position. Tighten all machine screws securely after alignment.

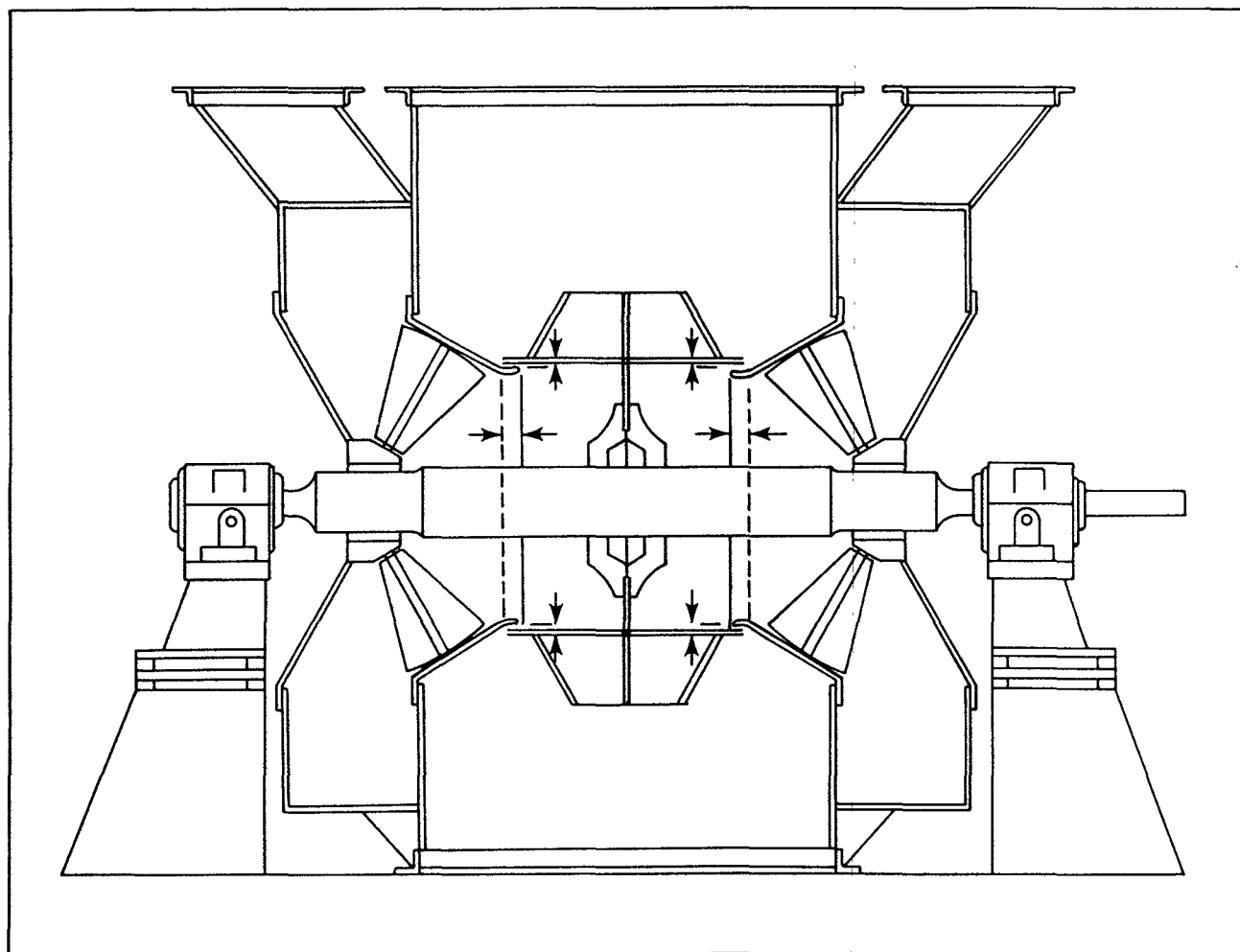


Fig. 5.5.4-13 Typical Cross Sectional View of Fan

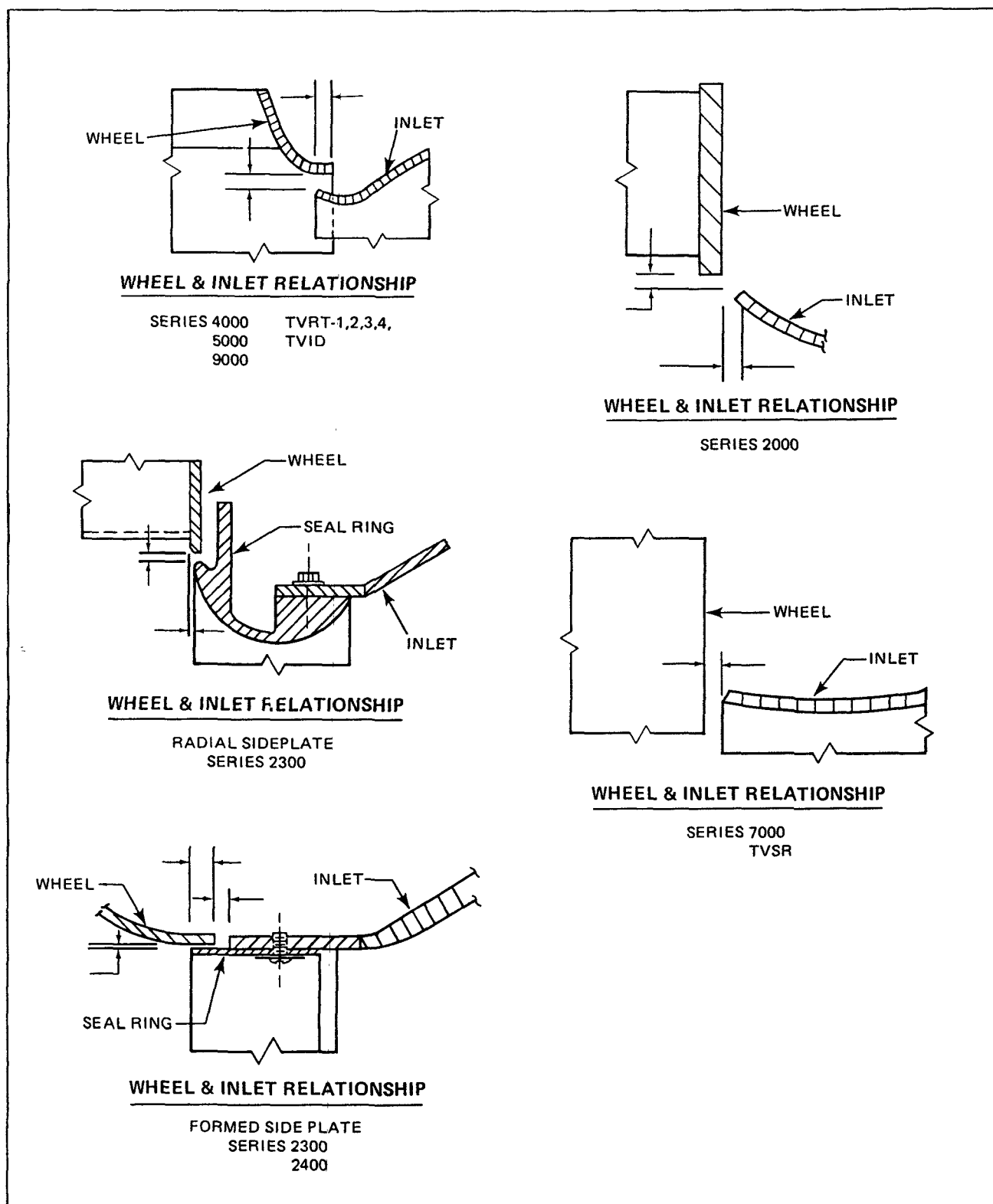


Fig. 5.5.4-14 Typical Wheel-Inlet Relationship Details

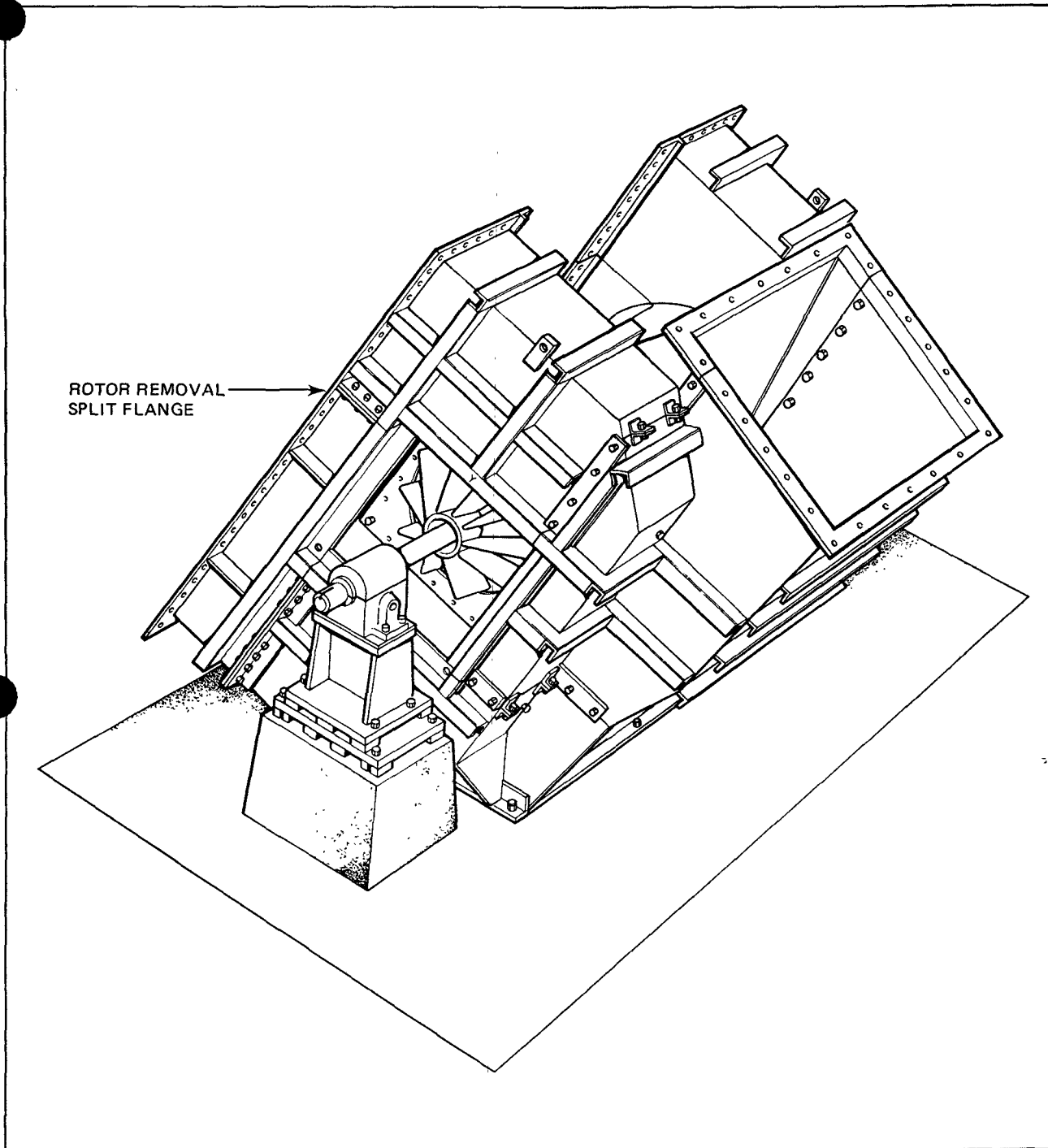


Fig. 5.5.4-11 Housing Construction - Type 4 Tenth Erection Stage

N. Place gasket material (if required) on Rotor Removal Split Flanges, and tie or tape temporarily. Lift Rotor Removal section in position. Use drift pins to align bolt holes in the mating angle iron flanges.

O. If sections do not line up, check assembly for misaligned sections installed previously. The Housing/Box

sections were factory-mated and match marked. If all sections are correctly aligned they will mate properly. Correct any misaligned sections.

P. Install and tighten all bolts and nuts in Rotor Removal Section Flanges. Refer to Torque Values in Section 5.10.

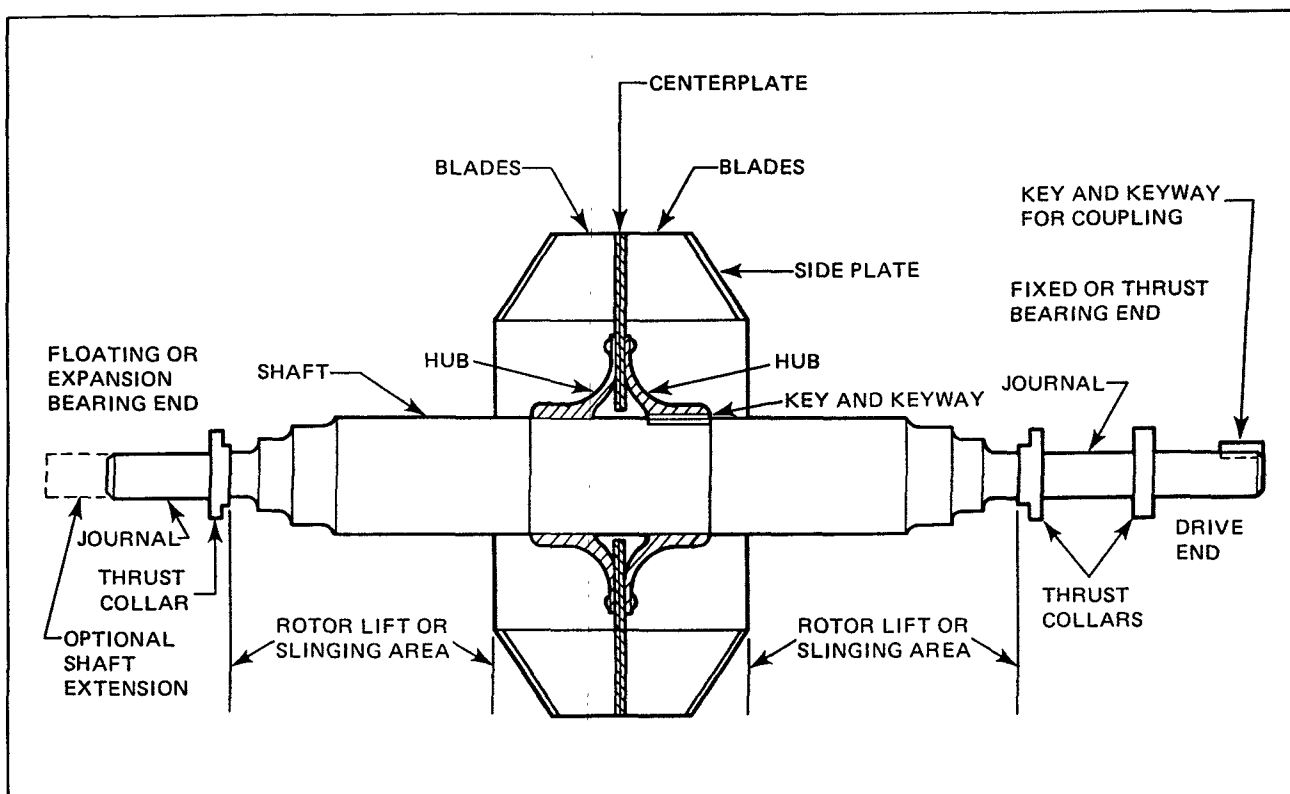


Fig. 5.7.1-2 Typical Rotor Component Identification

determined from the Main Drive end as specified by customer. See Figure 5.7.1-3 for Rotation and Wheel Blade Types.

6. Identify and move Fan Inlets or Vane and Inlet Assemblies to their respective Rotor ends.

- Inlets without Vanes are interchangeable and can be assembled to either side of Double Width-Double Inlet Fan.
- Vane and Inlet Assemblies are NON-interchangeable, and must be installed on the correct side of fan. Refer to Section 8 - Inlet Vane Controls, Preparation for Installation.

NOTE

Refer to Contract/Main Assembly Drawing for position of Vane Operating Lever. Vane Operating Lever must be in that approximate position when Vane and Inlet Assemblies are placed on Rotor.

7. Guide Inlets or Vane and Inlet Assemblies on Shaft. Place in approximate installation position; support securely to prevent damage to equipment or injury to personnel.

January 1978

NOTE

Assure that all Non-Split Inlet Parts, such as Inlet Vane Control Seal Parts, are placed on Shaft in proper installation sequence.

8. Pressurized Air Shaft Seals Only

Refer to Contract/Main Assembly Drawing, if Pressurized Air Shaft Seal Assembly is listed in Bill of Material; refer to Section 11.3 for seal details. (If not listed, proceed to Step 9.)

Seal Bodies are to be placed on the Rotor in their respective position to their location on the fans.

CAUTION

SEAL BODIES HAVE MACHINED SURFACES AND MUST BE HANDLED WITH CARE.

Insert a minimum of four (4) wooden wedges between machined Seal Body inside diameter and Shaft outside diameter and secure into position.

9. Install Bearings on Rotor. Refer to Section 6 - Bearings.

Section 9.0 Inlet Dampers



I.B. 90-400-9.0.0.1

GENERAL

Inlet Dampers are designed to be installed on the Inlet Flanges of the Fan Inlet Boxes, and to spin the air or gas stream in the direction of Wheel rotation when the Dampers are partially open, as indicated in Figure 9.0-1.

The Damper Shafts are offset in the Leafs so that the Leafs extend approximately 55% and 45% on either side of the Shaft centerline as shown in Figure 9.0-2.

When in the open position, the longer portion of the Leaf is toward the Fan, and the shorter portion is toward the duct or air entering side of the Damper. The Damper Leafs will extend beyond their channel frame when in the

open position, and therefore extend into the Fan Inlet Box and the connecting duct work.

The Dampers are marked with a welded "X" or "Y" on the inside upper surface of the channel flange to be assembled to the Inlet Box flange, with the corresponding welded "X" or "Y" as shown in Figure 9.0-3.

"X" indicates clockwise air spin rotation, and "Y" counterclockwise, when viewed from the specific inlet - NOT the Fan rotation designation. These are NOT alignment match marks, and need not be in line.

SWSI Fans have only one (1) Inlet Damper, and therefore the Drive Shaft extends beyond the Damper

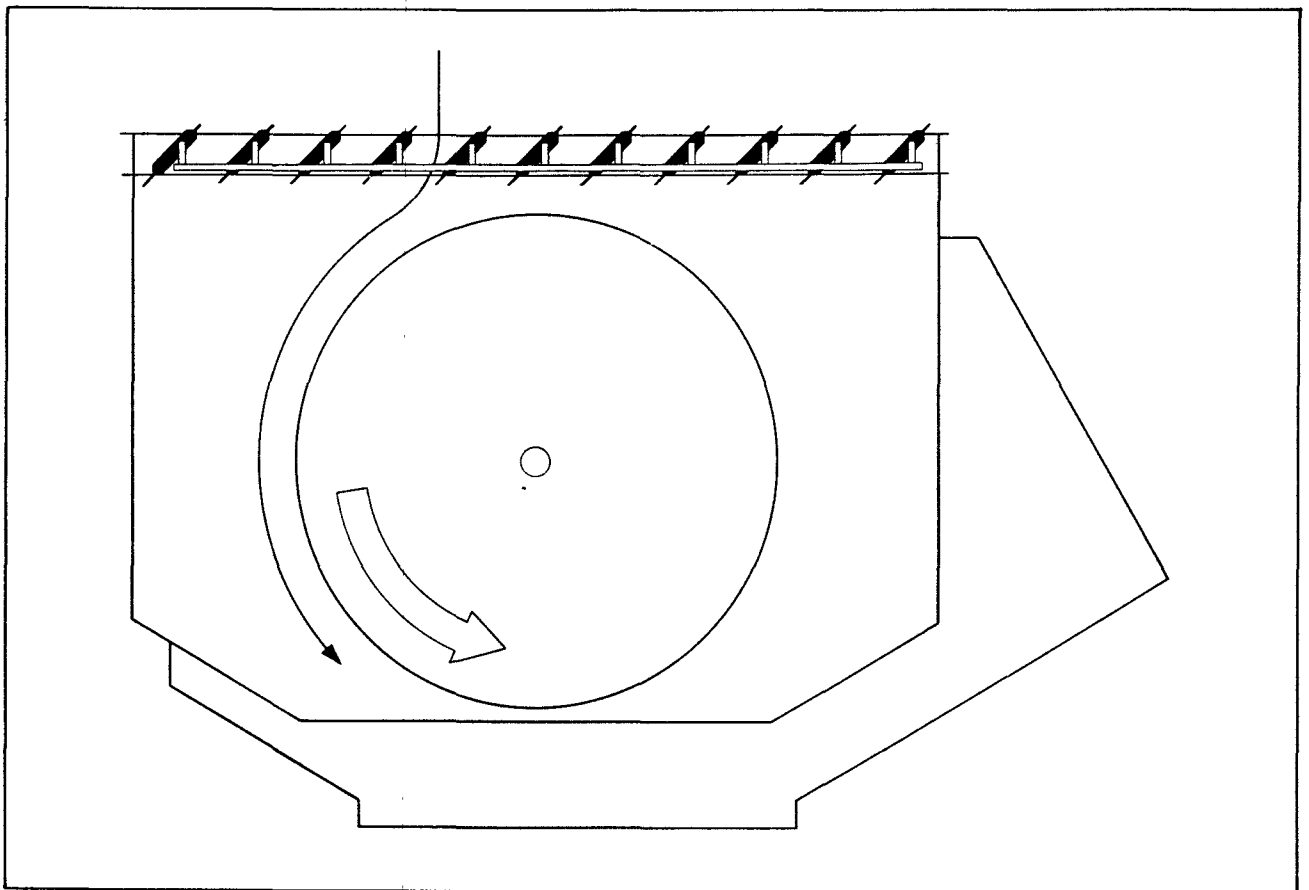


Fig. 9.0-1 *Inlet Damper Orientation*

Effective January 1978

Westinghouse Electric Corporation
Sturtevant Division
Hyde Park, Boston, MA 02136

IP7_039138

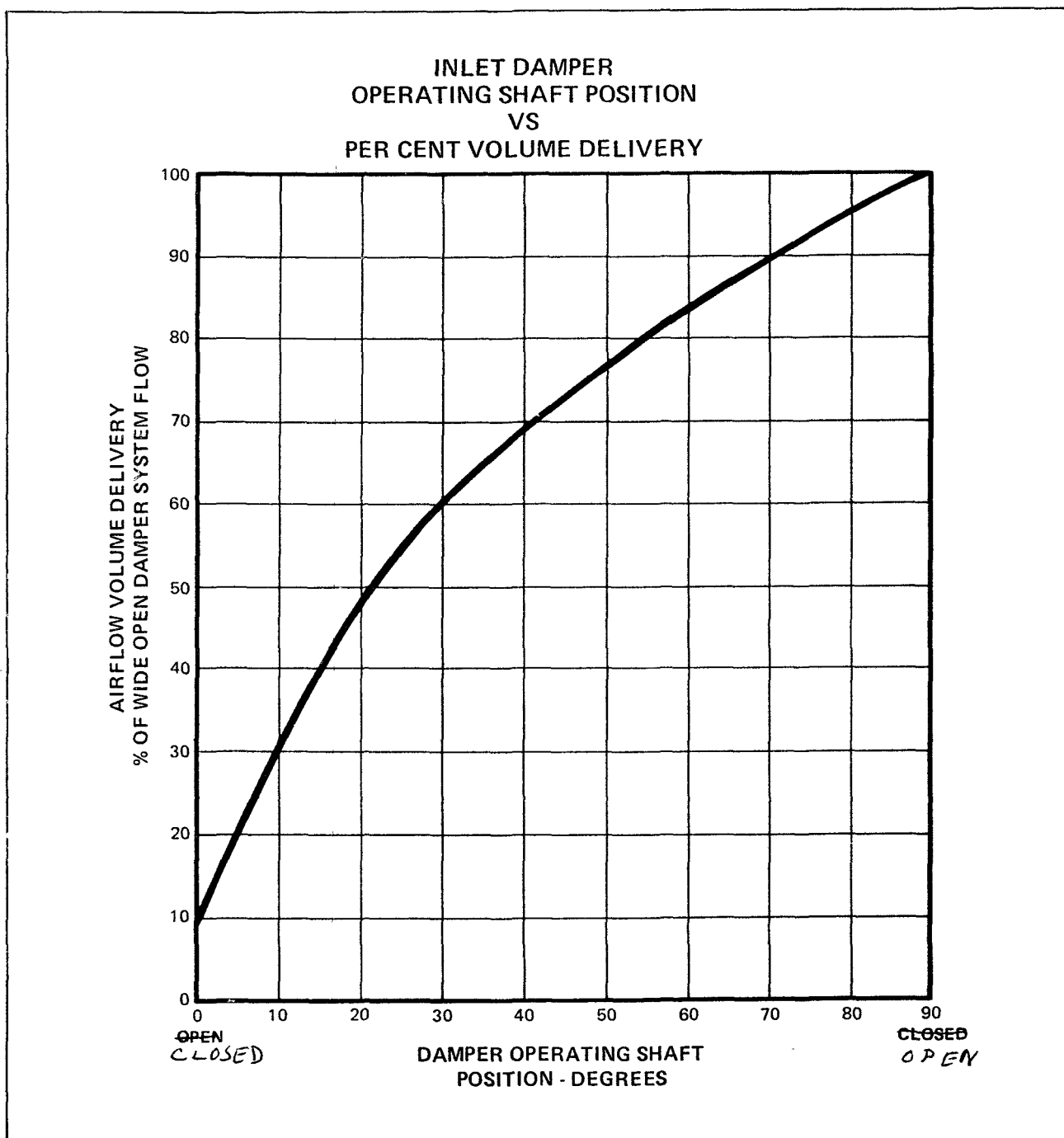


Fig. 9.1-3 *Inlet Damper Flow Curve*

See solution on next page

COST/BENEFIT ANALYSIS

CASE STUDY

You are a Senior Systems Analyst leading the feasibility phase for analysing a purchasing/order processing/inventory control system for Twentieth Century Frox. A system has been proposed which appears to meet the users' needs. There is no real "system" in place today. The expected life of the proposed system is three years. Twentieth Century Frox projects a 15% company growth compounded annually.

Your mandate is to produce a cost/benefit analysis of the proposed system and to provide recommendations to the senior management of Twentieth Century Frox.

Here are some other facts to aid you in reaching a solution.

- Another project team, responsible for systems development, has provided their initial estimates:
 - \$230,000.00
 - 6 months
- Upon investigation you find this team experienced an average 85-100% cost/time overrun on their last three projects. However, the quality of the end product is generally high. Development is scheduled to begin in two months.
- The proposed system is purported to have the following benefits:
 - A. Overtime (currently \$150,000 per year) would be reduced by 10%.
 - B. Inventory carrying costs would be reduced by \$250,000.
 - C. Profit on investment from inventory reduction would be \$125,000.
 - D. Anticipated profit on the reduction of lost sales is \$32,000 based on sales and order desk statistics.
 - E. Estimated operating cost of the new system (including hardware rental and software licensing) is \$14,000 a year.